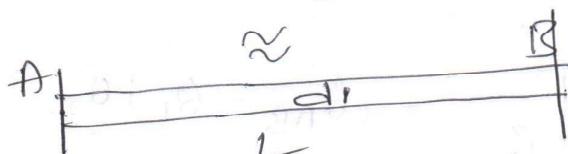
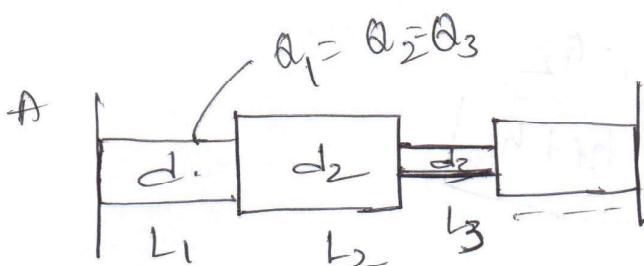


Dupit's Equation:-

$$h_f AB \text{ eff} = h_f AB \text{ Ind}$$

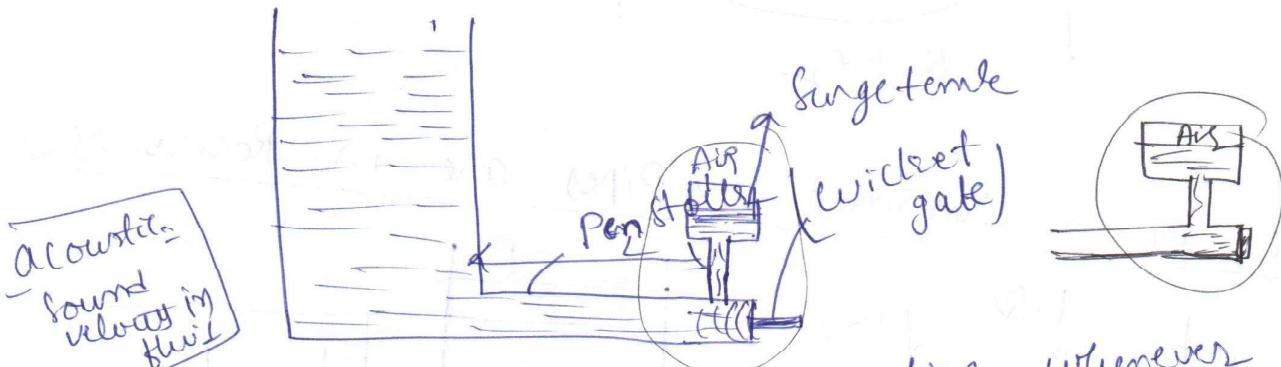


$$h_f + h_{f2} + h_{f3} = h_f$$

$$\frac{F \cdot L_1 d_1^2}{12 \cdot 1 d_1^5} + \frac{F \cdot L_2 d_2^2}{12 \cdot 1 d_2^5} + \dots = \frac{F \cdot L d^2}{12 \cdot 1 d^5}$$

$$\frac{L}{d^5} = \frac{L_1}{d_1^5} + \frac{L_2}{d_2^5} + \dots$$

Hammering effect :- \rightarrow closure valve surge tank



\rightarrow In flow through a pipe line whenever valve is closed because of the disturbance momentum of flow fluid a pressure wave will be generated and travels in opposite direction with acoustic speed by hitting the walls known as hammering effect.

\rightarrow To avoid these effect surge tanks will be used in the penstocks.

$$\text{Hammering effect vel} = C = \sqrt{\frac{A}{\rho}} \rightarrow \text{liquid}$$

$$= \sqrt{\gamma RT} \rightarrow \text{gas}$$

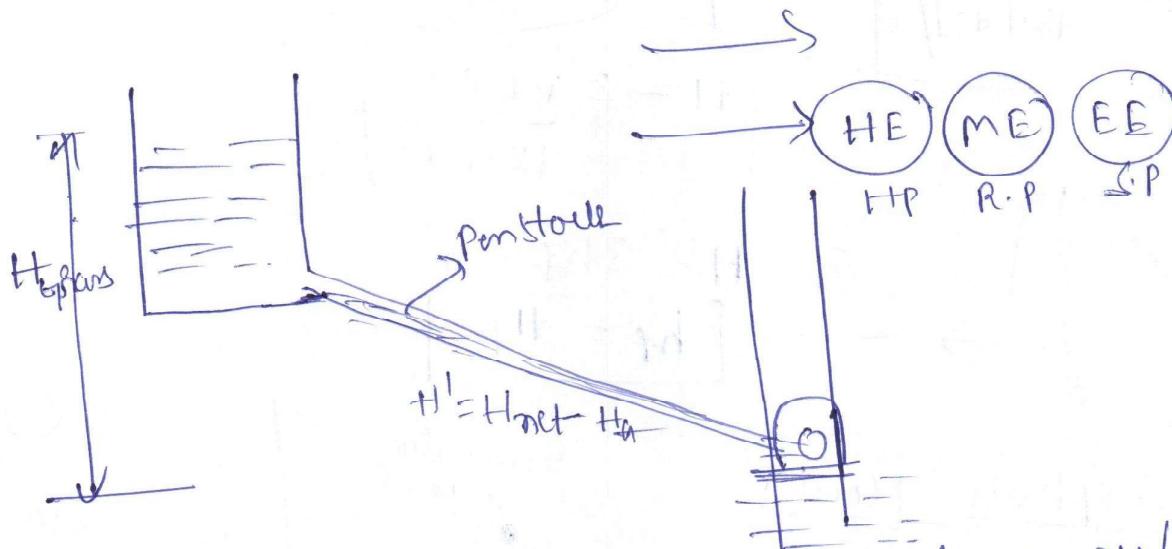
→ sudden closure $t < \frac{2L}{C}$
 't' → time of closure

→ gradual closure $t > \frac{2L}{C}$

Alleij's equation $P = \rho C v$

→ Power transmission :-

→ Net hydraulic power / water power



→ Net available power (δ) / hydraulic power / water power (P) = $\gamma \alpha (H - h_f)$

$$W = N \cdot \frac{\rho g}{\mu} \cdot \frac{M^2}{sec} \cdot M = N \cdot M \cdot \frac{g}{\mu sec}$$

$$P = \gamma \alpha (H - h_f)$$

Loss in power
 Frictional
 pumping

$$\eta = \frac{\gamma \alpha (H - h_f)}{\gamma \alpha H} = \left(\frac{H - h_f}{H} \right)$$

$$\eta = \frac{r \alpha (H - h_f)}{r \alpha H} = \frac{(H - h_f)}{H}$$

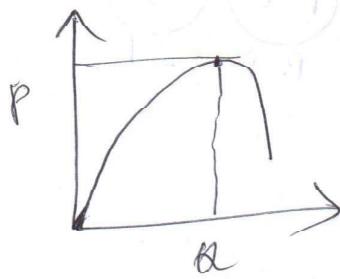
* → Condition for maximum power :-

$$h_f = \frac{H}{3}$$

$$P_E r \alpha t (H - h_f) = r \alpha \left[H - \frac{F L Q^2}{12 \cdot I d^5} \right]$$

$$\frac{dP}{da} = 0 \Rightarrow \frac{d}{da} \left[r \left[H - \frac{F L Q^2}{12 \cdot I d^5} \right] \right] = 0$$

$$\Rightarrow \frac{d}{da} \left[r \left[H - \frac{F L Q^2}{12 \cdot I d^5} \right] \right] = 0 = r \left[H - \frac{3 F L Q^2}{12 \cdot I d^5} \right] = 0$$

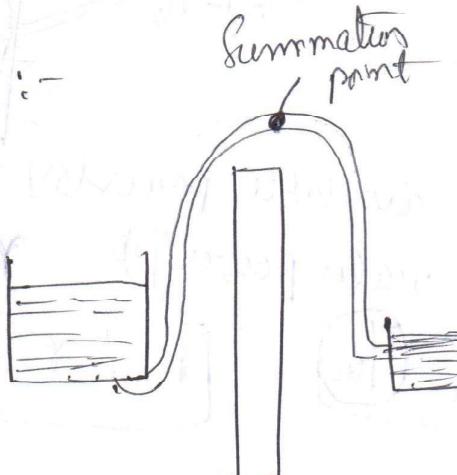
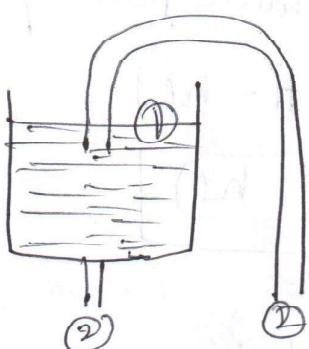


$$\left[H - \frac{3 F L Q^2}{12 \cdot I d^5} \right] = 0$$

$$H - 3 h_f = 0$$

$$h_f = \frac{H}{3}$$

* → Syphon flow :-



$$\begin{aligned} \eta_{min} &= \left(\frac{H - H/3}{H} \right) \\ &= 2/3 \times 100 \\ \eta &= 66.7\% \end{aligned}$$

$$\text{Saturation pressure} \rightarrow P_s = P_{atm} \Rightarrow P_{vap}$$

$$P_s = P_{atm} \Rightarrow P_{vap}$$

G-13

1) A pipe line connecting two reservoirs 200 mm dia and 2 km length. Friction factor = 0.04 with head loss of 8 m according for frictional entry and exit losses. The approximate velocity flow through pipes

Q:

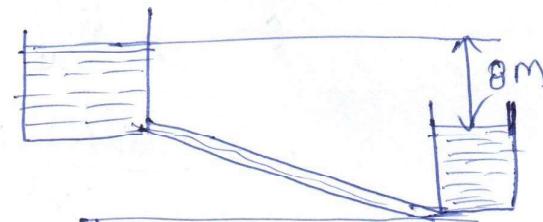
200 mm dia
2 km length

$$F = 0.04 \checkmark$$

$$z_2 = 8 \text{ m} \checkmark$$

Frictional, Entry & exit losses

Vel.?



$$E_1 = E_2 + h_{loss}$$

$$\left[z_1 + \frac{P_1}{\rho g} + \frac{V_1^2}{2g} \right] = \left[z_2 + \frac{P_2}{\rho g} + \frac{V_2^2}{2g} \right] + [h_f + h_{entry} + h_{exit}]$$

$$z_1 - z_2 = \frac{FLV^2}{2gd} + 0.5 \frac{V^2}{2g} + \frac{V^2}{2g} = 8 + 0.5 \frac{V^2}{2g} + \frac{V^2}{2g} = 8$$

$$8 = \frac{V^2}{2g} \left[\frac{FL}{d} + 0.5 + 1.0 \right]$$

$$V = \sqrt{\frac{2 \times 9.81 \times 8}{0.04 \times 2000 + 1.5}} = 0.62 \text{ m/s}$$

2) water at

G-09

$\rightarrow 25^\circ \text{C}$

upto 200 mm dia

$f = 0.02$, and pumping power required = ?

Q:

$\rightarrow 1 \text{ km}$

$\frac{Q = 0.07 \text{ m}^3/\text{sec}}{\text{& } 200 \text{ mm dia}}$

$\rightarrow f = 0.02$

pumping power = ? $\frac{\gamma Q hf}{12.1 d^3} = 9810 \times 0.07 \times 0.02 \times 10^3 \times 0.07^2$

$$\text{where } hf = \frac{FLV^2}{2gd} \& \frac{EV^2}{12.1 d^3} \left(= 17.4 \text{ kN} \right)$$

⑧ The Hammering velocity of flow water through rigid pipe $k_w = 1.96 \text{ GPa}$

$$k_w = 1.96 \text{ GPa}$$

$$c = \sqrt{k_w / \rho}$$

$$= \frac{19.6 \times 10^9}{10^3}$$

$$= 1400 \text{ m/s}$$

Hammering effect $c = \sqrt{k_w / \rho}$

⑨ The ~~turbine~~ connected with penstock of length 3 km. the pressure wave travels with velocity 1500 m/sec. The turbine gates are closed uniformly and completely in a period 4.5 sec if it is called gradual

$$L = 3 \text{ km length}$$

$$c = 1500 \text{ m/s} \quad t > \left(\frac{2L}{c} \right)$$

$$t = 4.5 \text{ sec} \quad 4.5 = \frac{2 \times 3000}{1500}$$

$$4.5 > 4$$

⑩ A piping system consists of 3 pipes arranged in series whose lengths are 1200 & 750, 600, 450 mm dia 750, 600, 450 mm transform the system into an equally ? (equal length) dia of pipe 450 mm

$$L = 1200, 750, 600$$

$$\text{dia} = 750, 600, 450$$

$$450$$

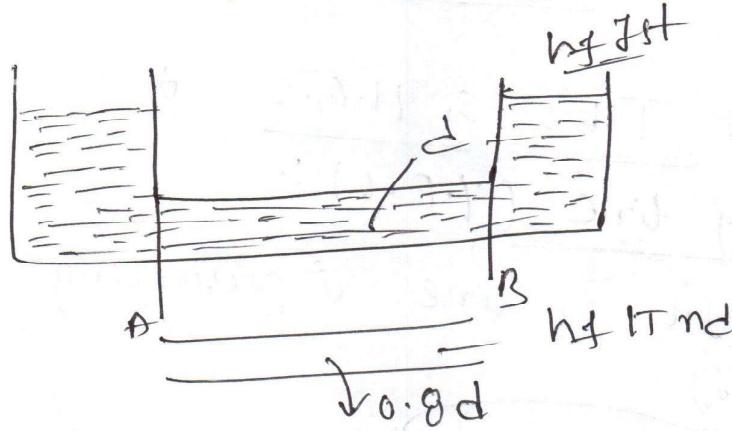
$$\frac{L}{d^5} = \frac{L_1}{d_1^5} + \frac{L_2}{d_2^5} + \frac{L_3}{d_3^5}$$

$$\frac{L_e}{(450)^5} = \frac{1200}{(750)^5} + \frac{750}{(600)^5} + \frac{600}{(450)^5}$$

$$L_e = \underline{\underline{871.3 \text{ m}}}$$

Q) A pipe line connecting two reservoirs have its diameter reduced by 20% due to deposition of chemicals with an altered (changed) friction factor. How would cause a reduction in flow rate of ?

Ans:



Reduction in flow rate ?

$$\frac{(\alpha_1 - \alpha_2)}{\alpha_1} \times 100$$

Given head difference :-

$$E_1 = E_2 + \cancel{h_f} = E_1 - E_2 = C$$

$$h_f_{AB_1} = h_f_{AB_2}$$

$$\frac{FL\alpha_1^2}{12.1 d^5} = \frac{FL\alpha_2^2}{12.1 d^5} \Rightarrow \frac{FL\alpha_1^2}{12.1 d^5} = \frac{FL\alpha_2^2}{12.1 (0.8d)^5}$$

$$\alpha_1^2 = \frac{\alpha_2^2}{(0.8)^5} \Rightarrow \alpha_2 = 0.8^{\frac{2.5}{5}} \alpha_1$$

$$\alpha_{\text{Reduction}\%} = \left[\frac{\alpha_1 - 0.8^{\frac{2.5}{5}} \alpha_1}{\alpha_1} \right] \times 100 = \underline{\underline{42.8\%}}$$

7) water flows through a circular pipe of 10cm dia at a velocity of 0.1 m/s kinematic viscosity $\nu_w = 10^{-5} \text{ m}^2/\text{sec}$ $F = \frac{1}{\text{Re}}$ (Friction factor)

sol -

$$\frac{d}{2} = 10 \text{ cm dia}$$

$$V = 0.1 \text{ m/sec}$$

$$\nu_w = 10^{-5} \text{ m}^2/\text{sec}$$

$$F = \frac{64}{\text{Re}} \Rightarrow F = \frac{64}{\frac{\rho V D}{\mu}}$$

$$f = 0.005 \text{ to } 0.01$$

$$F = f \frac{64}{\frac{\rho V D}{\mu}} = \frac{64}{\frac{(0.1) \times (0.1)}{10^{-5}}} \neq \frac{64}{1000}$$

$$F = 0.064$$

T.E.L $E_1 + H.G.L$

Total energy line (T.E.L) :-
 \rightarrow T.E.L is line representing the total available energy

$$\rightarrow + \frac{P}{\rho g} + \frac{V^2}{2g}$$

\rightarrow T.E.L will be horizontal for idle flow
~~but it always slopes downwards for real fluid flow~~

Hydraulic gradient line :- (H.G.L) (S)

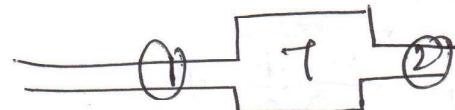
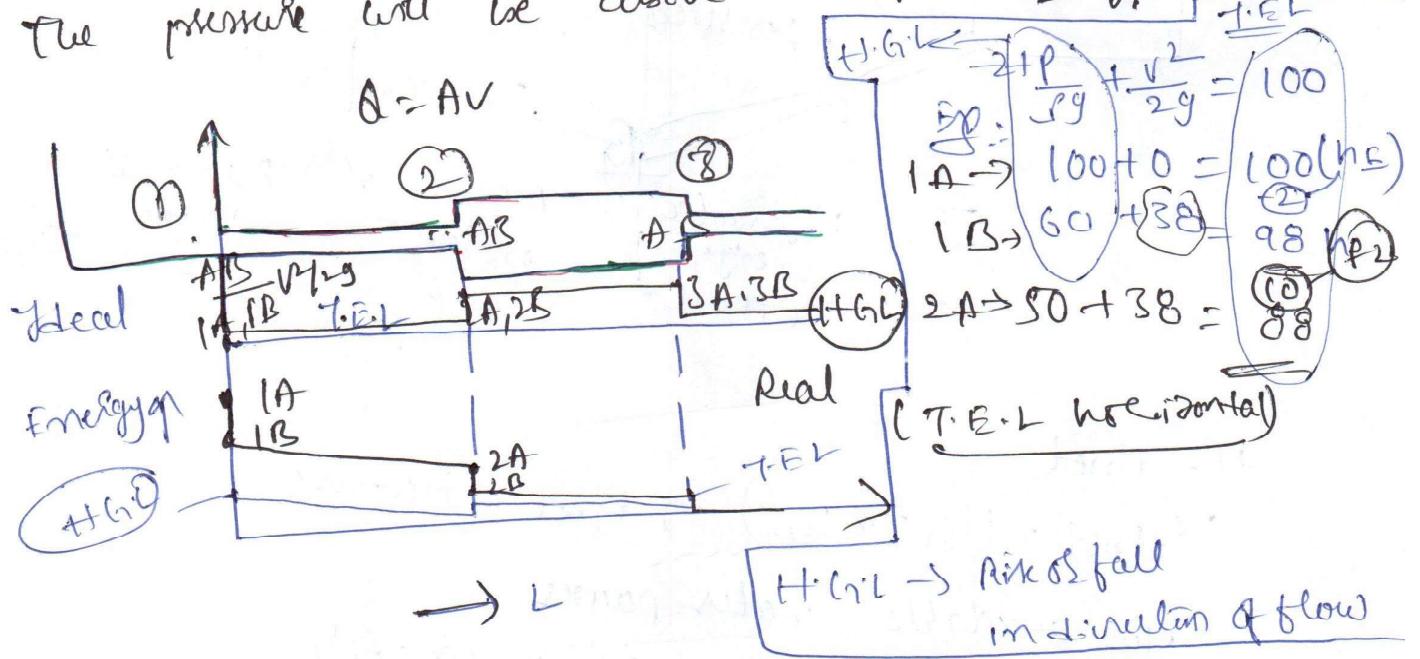
Piezometric line :-

\rightarrow A line representing the available piezometric head

$$\frac{Z + P}{\rho g}$$

$$\text{T.E.L} - \text{H.G.L} = \frac{V^2}{2g}$$

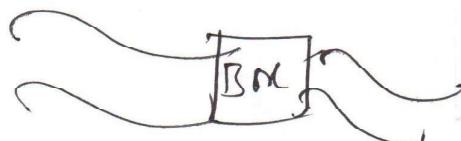
Note 5-1) $H \cdot g \cdot L$ may rise or fall in the direction of flow
 2) $H \cdot g \cdot L$ represents atm. pressure condition for all the points about $H \cdot g \cdot L$. The pressure will be below atmospheric for all the points below $H \cdot g \cdot L$ the pressure will be above atmospheric [Syphon flow].



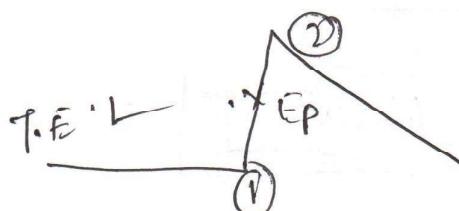
$$E - E_1 = E_2 + h_{loss}$$



$$E_1 + E_2 = E_2 + h_{loss}$$



$$P = \gamma Q H$$



$$E = ?$$

$$\text{Power} = P'$$

$$\frac{P}{\gamma Q}$$