$\mathbf{R05}$

Set No. 2

II B.Tech II Semester Examinations, APRIL 2011 THERMODYNAMICS Aeronautical Engineering

Time: 3 hours

Max Marks: 80

Answer any FIVE Questions All Questions carry equal marks *****

- 1. (a) Derive an expression for heat transfer in a non-flow constant volume process.
 - (b) A spherical shaped of 14 M diameter contain ${}^{\prime}H_{2}{}^{\prime}$ at 33^oC and 1.3 bar. Find the mass of ${}^{\prime}H_{2}{}^{\prime}$ in the balloon using real gas equation. [7+9]
- 2. In an ammonia vapour compression system, the pressure in the evaporator is 2 bar. Ammonia at exit is 0.85 dry and at entry its dryness fraction is 0.19. During compression, the work done per kg of ammonia is 150kJ. Calculate the C.O.P and volume of vapour entering the compressor per minute, if the rate of ammonia circulation is 4.5kg/min. the latent heat and specific volume at 2 bar are 1325kJ/kg and $0.58m^3$ /kg respectively. [16]
- 3. (a) Write the differences between system and control volume.
 - (b) A gas undergoes a reversible non-flow process according to the relation P = (-3V+15) where V is the volume in m³ and P is the pressure in bar. Determine the work done when the volume changes from 3 to 6 m³. [6+10]
- 4. A gas undergoes a thermodynamic cycle consisting of the following processes:
 - (a) Process 1-2: Constant pressure, P=1.4 bar, V1 = 0.028 m³, W₁₂ = 10.5 KJ,
 - (b) Process 2-3 compression with PV = constant, $U_3 = U_2$,
 - (c) Process 3-1: Constant volume, $U_1 U_3 = -26.4$ KJ.

- (a) sketch the cycle on a P-V diagram
- (b) Calculate the net work for the cycle in KJ
- (c) calculate the heat transfer for process 1-2
- (d) show that $\oint \delta Q = \oint \delta W$. [16]
- 5. Draw T-s and p-V diagrams of a Otto cycle. Derive an expression for the air standard efficiency of Otto cycle in terms of compression ratio and net work down per kg of air. [16]
- 6. (a) Explain the influence of pressure on the following properties of steam preferably with the help of T-S diagram. 1. Enthalpy of evaporation 2. Enthalpy of dry saturated steam.
 - (b) 5kg of steam with a dryness fraction of 0.9 expand adiabatically according to the law $PV^{1.13} = C$ from a pressure of 8 bar to 1.5 bar. Determine 1. Final dryness fraction 2. Heat transferred 3. Work done. [7+9]
- 7. (a) Explain

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- i. Dry bulb temperature
- ii. Wet bulb depression.
- (b) Air $(N_2 = 77\%, O_2 = 23\% by weight) at 27^{\circ}$ C and 13 bar is contained in a vessel of capacity $0.6m^3$. Some quantity of CO_2 is forced into the vessel so that the temperature remains at 27° C but the pressure rises to 19 bar. Find the masses of $O_2, N_2 and CO_2$ in the cylinder. [6+10]
- 8. Two reversible heat engines A and B are arranged in series. Engine A rejects heat directly to engine B. A receives 200 KJ at a temperature of 421^oC from the hot source while engine B is in communication with a cold sink at a temperature of 5^oC. If the work output of A is twice that of B find
 - (a) intermediate temperature between A and B and
 - (b) efficiency of each engine.

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Set No. 4

II B.Tech II Semester Examinations, APRIL 2011 THERMODYNAMICS Aeronautical Engineering

Time: 3 hours

Max Marks: 80

Answer any FIVE Questions All Questions carry equal marks *****

- 1. (a) Explain
 - i. Dry bulb temperature
 - ii. Wet bulb depression.
 - (b) Air $(N_2 = 77\%, O_2 = 23\% by weight) at 27^{\circ} C$ and 13 bar is contained in a vessel of capacity $0.6m^3$. Some quantity of CO_2 is forced into the vessel so that the temperature remains at $27^{\circ} C$ but the pressure rises to 19 bar. Find the masses of $O_2, N_2 and CO_2$ in the cylinder. [6+10]
- 2. (a) Derive an expression for heat transfer in a non-flow constant volume process.
 - (b) A spherical shaped of 14 M diameter contain ${}^{\prime}H_{2}{}^{\prime}$ at 33^oC and 1.3 bar. Find the mass of ${}^{\prime}H_{2}{}^{\prime}$ in the balloon using real gas equation. [7+9]
- 3. Two reversible heat engines A and B are arranged in series. Engine A rejects heat directly to engine B. A receives 200 KJ at a temperature of 421°C from the hot source while engine B is in communication with a cold sink at a temperature of 5°C. If the work output of A is twice that of B find
 - (a) intermediate temperature between A and B and
 - (b) efficiency of each engine.

[16]

[16]

- 4. (a) Write the differences between system and control volume.
 - (b) A gas undergoes a reversible non-flow process according to the relation P = (-3V+15) where V is the volume in m³ and P is the pressure in bar. Determine the work done when the volume changes from 3 to 6 m³. [6+10]
- 5. A gas undergoes a thermodynamic cycle consisting of the following processes:
 - (a) Process 1-2: Constant pressure, P=1.4 bar, V1 = 0.028 m³, $W_{12} = 10.5$ KJ,
 - (b) Process 2-3 compression with PV = constant, $U_3 = U_2$,
 - (c) Process 3-1: Constant volume, $U_1 U_3 = -26.4$ KJ.

- (a) sketch the cycle on a P-V diagram
- (b) Calculate the net work for the cycle in KJ
- (c) calculate the heat transfer for process 1-2
- (d) show that $\oint \delta Q = \oint \delta W$.

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- 6. (a) Explain the influence of pressure on the following properties of steam preferably with the help of T-S diagram. 1. Enthalpy of evaporation 2. Enthalpy of dry saturated steam.
 - (b) 5kg of steam with a dryness fraction of 0.9 expand adiabatically according to the law $PV^{1.13} = C$ from a pressure of 8 bar to 1.5 bar. Determine 1. Final dryness fraction 2. Heat transferred 3. Work done. [7+9]
- 7. In an ammonia vapour compression system, the pressure in the evaporator is 2 bar. Ammonia at exit is 0.85 dry and at entry its dryness fraction is 0.19. During compression, the work done per kg of ammonia is 150kJ. Calculate the C.O.P and volume of vapour entering the compressor per minute, if the rate of ammonia circulation is 4.5kg/min. the latent heat and specific volume at 2 bar are 1325kJ/kg and 0.58m³/kg respectively. [16]
- 8. Draw T-s and p-V diagrams of a Otto cycle. Derive an expression for the air standard efficiency of Otto cycle in terms of compression ratio and net work down per kg of air. [16]

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Set No. 1

II B.Tech II Semester Examinations, APRIL 2011 THERMODYNAMICS Aeronautical Engineering

Time: 3 hours

Max Marks: 80 Questions

Answer any FIVE Questions All Questions carry equal marks *****

- 1. (a) Explain
 - i. Dry bulb temperature
 - ii. Wet bulb depression.
 - (b) Air $(N_2 = 77\%, O_2 = 23\% by weight) at 27^{\circ} C$ and 13 bar is contained in a vessel of capacity $0.6m^3$. Some quantity of CO_2 is forced into the vessel so that the temperature remains at $27^{\circ} C$ but the pressure rises to 19 bar. Find the masses of $O_2, N_2 and CO_2$ in the cylinder. [6+10]
- 2. (a) Derive an expression for heat transfer in a non-flow constant volume process.
 - (b) A spherical shaped of 14 M diameter contain ${}^{\prime}H_{2}{}^{\prime}$ at 33^oC and 1.3 bar. Find the mass of ${}^{\prime}H_{2}{}^{\prime}$ in the balloon using real gas equation. [7+9]
- 3. A gas undergoes a thermodynamic cycle consisting of the following processes:
 - (a) Process 1-2: Constant pressure, P=1.4 bar, V1 = 0.028 m³, $W_{12} = 10.5$ KJ,
 - (b) Process 2-3 compression with PV = constant, $U_3 = U_2$,
 - (c) Process 3-1: Constant volume, $U_1 U_3 = -26.4$ KJ.

- (a) sketch the cycle on a P-V diagram
- (b) Calculate the net work for the cycle in KJ
- (c) calculate the heat transfer for process 1-2
- (d) show that $\oint \delta Q = \oint \delta W$. [16]
- 4. (a) Explain the influence of pressure on the following properties of steam preferably with the help of T-S diagram. 1. Enthalpy of evaporation 2. Enthalpy of dry saturated steam.
 - (b) 5kg of steam with a dryness fraction of 0.9 expand adiabatically according to the law $PV^{1.13} = C$ from a pressure of 8 bar to 1.5 bar. Determine 1. Final dryness fraction 2. Heat transferred 3. Work done. [7+9]
- 5. In an ammonia vapour compression system, the pressure in the evaporator is 2 bar. Ammonia at exit is 0.85 dry and at entry its dryness fraction is 0.19. During compression, the work done per kg of ammonia is 150kJ. Calculate the C.O.P and volume of vapour entering the compressor per minute, if the rate of ammonia circulation is 4.5kg/min. the latent heat and specific volume at 2 bar are 1325kJ/kg and $0.58m^3$ /kg respectively. [16]
- 6. (a) Write the differences between system and control volume.

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- (b) A gas undergoes a reversible non-flow process according to the relation P = (-3V+15) where V is the volume in m³ and P is the pressure in bar. Determine the work done when the volume changes from 3 to 6 m³. [6+10]
- 7. Two reversible heat engines A and B are arranged in series. Engine A rejects heat directly to engine B. A receives 200 KJ at a temperature of 421°C from the hot source while engine B is in communication with a cold sink at a temperature of 5°C. If the work output of A is twice that of B find
 - (a) intermediate temperature between A and B and
 - (b) efficiency of each engine. [16]
- 8. Draw T-s and p-V diagrams of a Otto cycle. Derive an expression for the air standard efficiency of Otto cycle in terms of compression ratio and net work down per kg of air. [16]

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Set No. 3

II B.Tech II Semester Examinations, APRIL 2011 THERMODYNAMICS Aeronautical Engineering

Time: 3 hours

Max Marks: 80

Answer any FIVE Questions All Questions carry equal marks *****

- 1. In an ammonia vapour compression system, the pressure in the evaporator is 2 bar. Ammonia at exit is 0.85 dry and at entry its dryness fraction is 0.19. During compression, the work done per kg of ammonia is 150kJ. Calculate the C.O.P and volume of vapour entering the compressor per minute, if the rate of ammonia circulation is 4.5kg/min. the latent heat and specific volume at 2 bar are 1325kJ/kg and $0.58m^3$ /kg respectively. [16]
- 2. (a) Derive an expression for heat transfer in a non-flow constant volume process.
 - (b) A spherical shaped of 14 M diameter contain ${}^{\prime}H_{2}{}^{\prime}$ at 33^oC and 1.3 bar. Find the mass of ${}^{\prime}H_{2}{}^{\prime}$ in the balloon using real gas equation. [7+9]
- Draw T-s and p-V diagrams of a Otto cycle. Derive an expression for the air standard efficiency of Otto cycle in terms of compression ratio and net work down per kg of air. [16]
- 4. Two reversible heat engines A and B are arranged in series. Engine A rejects heat directly to engine B. A receives 200 KJ at a temperature of 421°C from the hot source while engine B is in communication with a cold sink at a temperature of 5°C. If the work output of A is twice that of B find
 - (a) intermediate temperature between A and B and
 - (b) efficiency of each engine.

[16]

[16]

- 5. A gas undergoes a thermodynamic cycle consisting of the following processes:
 - (a) Process 1-2: Constant pressure, P=1.4 bar, V1 = 0.028 m³, $W_{12} = 10.5$ KJ,
 - (b) Process 2-3 compression with PV = constant, $U_3 = U_2$,
 - (c) Process 3-1: Constant volume, $U_1 U_3 = -26.4$ KJ.

- (a) sketch the cycle on a P-V diagram
- (b) Calculate the net work for the cycle in KJ
- (c) calculate the heat transfer for process 1-2
- (d) show that $\oint \delta Q = \oint \delta W$.
- 6. (a) Write the differences between system and control volume.

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Set No. 3

- (b) A gas undergoes a reversible non-flow process according to the relation P = (-3V+15) where V is the volume in m³ and P is the pressure in bar. Determine the work done when the volume changes from 3 to 6 m³. [6+10]
- 7. (a) Explain the influence of pressure on the following properties of steam preferably with the help of T-S diagram. 1. Enthalpy of evaporation 2. Enthalpy of dry saturated steam.
 - (b) 5kg of steam with a dryness fraction of 0.9 expand adiabatically according to the law $PV^{1.13} = C$ from a pressure of 8 bar to 1.5 bar. Determine 1. Final dryness fraction 2. Heat transferred 3. Work done. [7+9]
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